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(54) **A DEVICE FOR PERPENDICULAR STRATIFICATION OF PLANARY FIBROUS SHAPES**

VORRICHTUNG ZUM SENKRECHT LAMINIEREN VON FLÄCHIGEN FASERIGEN FORMEN

DISPOSITIF SERVANT A EFFECTUER LA STRATIFICATION PERPENDICULAIRE DE FORMES
FIBREUSES PLANES

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GB-A- 2 222 835 **US-A- 2 638 959**
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Description

[0001] The scope of the patent is a device for production of voluminous shapes by perpendicular stratification of planary fibrous shapes, above all of fibrous web from a carding machine.

Description of the Prior Art

[0002] For the production of webbing from a web as obtained from machines with carding effect substantially three basic principles are known and used, based on parallel, cross or perpendicular web stratification. The method of the fibrous layer preparation and the related fibre orientation in the layer has a deciding importance for the product properties. In the case of voluminous products, which are exposed in their application to a single, repeated or longlasting stress, the best properties are obtained by preparing the fibrous layer by perpendicular stratification of the web. Under most variable conditions such products best retain their functional properties, especially as fillers or thermal insulants.

[0003] Several types of equipment working on rotational or vibration principle are known in the production of a fibrous layer composed of fibres laid substantially perpendicularly to the product plane.

[0004] Perpendicular laying devices on rotational principle form the web by means of various types of rotating elements such as gear wheels, cylinders with pins or rotating disks with specially shaped indents between which the web is fed. A merit of such systems protected e.g. by CZ AO 273997 is their high performance and a wide range of perpendicularly laid produced webbing. A limitation is their limited possibility of controlling the fibre position at various specific densities of the web, and a heterogenous structure of product surface. A deviation of fibre orientation from a perpendicular orientation to the layer surface makes the compressing resistance of the product decline. Rotating elements of the laying device such as e.g. a system of wires, formed disks or indentations produce a row-like structure, connected with an irregular density of fibre distribution in the product area.

[0005] Vibration laying devices work on the principle of shaping the fed web by a forming batten with an eventual upholding of shaped plaits by a thrust batten. The machines working on this principle are known in various arrangements, and are mostly adapted to the requirements on the product properties. So e.g. according to the US Patent 2.638960 a device consists of a horizontally vibrating cheek which in compressed condition feeds the web or a yarn system onto a base layer and in the back position the thrust batten presses the material in the form of a loop to the base layer with an adhesive.

[0006] According to CZ P 37 619 the web fed horizontally is bent by vertically situated battens from upside and from downside. The web fibres are attached from

upside as well as from downside to the base layers. CZ P 56 029 describes a device in vertical arrangement, in which the fed web is plaited between a pair of conveyor belts by a pair of battens rocking in reciprocating movements against each other. By attachment of adhesive-coated base layer fabrics fed from both sides a configuration resembling a double plush is formed. A kind of modification thereof is a device described in CZ P 87 556, in which the web fed from upside is taken over and deposited on an adhesive-coated fabric by a pair of alternately working battens in the form of a doffer comb. According to CZ AO 235494 the web from the carding machine is formed on the base directly on the doffing point at the outlet drum of the carder. In this method the positions of the formed plaits are not fixed, so that the product can hardly be transferred to the equipment for chemical or mechanical stiffening. All described mechanisms require to attach the perpendicularly shaped webbing to the base fabric by an adhesive immediately when forming individual plaits, otherwise the structure shaped here is unstable and does not permit to apply known methods of mechanical or adhesion stiffening.

[0007] A certain progress in this trend was brought in a device according to CZ AO 269 300 and the related Patent Application PV 1819-92. The web fed to the device from upside is formed in plaits by a vibrating fly comb and individual plaits are pressed to a fibrous layer built up between a conveyor belt and a grid by a synchronously vibrating batten.

[0008] The driving mechanism of the shaping elements which must exert a highly demanding and accurate motion, is solved according to PV 1819-92 by a four-joint assembly, featuring two groups of tie-rods driving the two vibrating elements - the fly comb and the batten. The light functional vibrating elements proper are directly connected to assemblies converting a circular motion to a reciprocating motion. A high stress of the assemblies converting the circular motion to a reciprocating motion generates shock forces and causes a vibration of working elements (fly comb, batten). Both shafts are interconnected by sets of three gear-wheels. Though the mechanism provides for a synchronous drive of both working elements, it permits, however, owing to a complicated setup and a considerable mass of the equipment, even in a carefully balanced condition an oscillation frequency merely up to 600 cycles per minute. Such capacity is far from satisfying the needs of modern carding machines, with which the laying device is incorporated into the production line. Said heavy mass of individual components according to this concept leads in a continuous uninterrupted operation to an accelerated wear of the gear-wheels, pins and bearings and consequently to a shorter life expectancy of the machine. With continuing operation time the noise level is gradually increased and the product becomes less uniform. Another consequence thereof is an uneven shaping of individual web plaits with an increasing frequency.

[0009] The vibrating elements must be coupled to-

gether with the prime mover by a number of joints, the distance from each other depending on the machine frequency required and on the bending rigidity of working elements. Any frequency increase requires therefore an increase in the number of joints and consequently of gear-wheels or of the mass of vibrating elements. Both said possibilities result in an increased overall mass of moving parts of the machine and make any further increase of the working frequency impossible.

Nature of the invention

[0010] The limitations mentioned above are remedied by a device for perpendicular stratification of planary fibrous shapes with two synchronously vibrating elements according to the invention. The vibrating elements are connected with the driving member indirectly by the intermediary of one or two robust shafts, rigidly fitted in bearings in a rigid machine framework. The vibrating elements are connected with the shaft(s) directly or by means of a set of flexible joints in sliding fitting. The flexible joints can consist of flat steel springs with a width-to-thickness ration higher than 10, but also of tie rods, having sliding fitting and flexible knuckle joints. The flexible joints with sliding fitting permit to convert a circular motion of the driving prime mover and of the massive shaft into a linear reciprocating motion of vibrating elements.

[0011] The mechanism can comprise a driving shaft with two crank assemblies fitted with a phase shift between each other. The shafts are driven by the driving mechanism synchronously so that they make a reciprocating motion around their longitudinal axes. Owing to its large diameter and high rigidity, the shaft (possibly a tube) transmitting a reciprocating motion, does not vibrate under the influence of the forces transmitted onto it by the driving mechanism. Thus the shaft transmits to the working element a motion which is evenly distributed along its whole width without generating any unwanted vibration. The merit of the device is in that it prevents any vibration of working elements even at the necessary high operating speed owing to the connection of said elements with rigidly seated shafts by means of a set of resilient joints, own mass of which is substantially smaller as compared with known transmission members. Such device is able to work at a frequency of 2000 cycles per minute. This is a frequency, permitting to process fibrous web with a speed corresponding to the speed of modern carding machines.

[0012] Other advantages of the device according to the invention are a high life expectancy at a minimum maintenance demand, a minimum of rotating parts, on which any fibres or web can be wound-on, an easy setting of back position of vibrating elements when adjusting the machine for processing various types of goods directly on the shaft tie-rods, a possibility of easy changing the amplitude of vibrating elements directly on the driving mechanism.

[0013] Setting of the phase shift for the vibration motion of the functional members is necessary for a safe web doffing from the doffer comb working edge. This enables an accurate plait shaping, which is a must for obtaining a smooth surface and an even product density.

Survey of figures in the drawings

[0014] Fig. 1 shows schematically a device for perpendicular stratification of planary fibrous shapes with one single shaft.

[0015] Fig. 2 shows schematically a device for perpendicular stratification of planary fibrous shapes with two shafts and one vibrating element in sliding fitting and a second vibrating element in rigid fitting.

[0016] Fig. 2a shows schematically a driving mechanism.

[0017] Fig. 3 shows schematically a device for perpendicular stratification of planary fibrous shapes with two shafts and vibrating elements in sliding fitting.

Examples of embodiment

Example 1

[0018] A device shown in Fig. 1 serves for processing of a fibrous layer, e.g. fibrous web coming from a carding machine. It consists of two vibrating elements 1 and 2 for perpendicular stratification of web. Said elements 1 and 2 are connected over a shaft 4 and connecting rod 10 with knuckle joint 11 and a driving mechanism 3. The shaft 4 makes a rotating and reciprocating motion along its longitudinal axis. The shaft 4 is a robust tube having an outer diameter 701 mm and is rigidly fitted in bearings in a rigid framework of the machine. The vibrating element 1 is rigidly connected with the shaft 4, the element 2 is joined by means of tie-rods 7 with sliding fitting and flexible knuckle joints.

[0019] The vibrating element 2 takes over the fibrous web by means of a set of needles from the element 1 and shapes a plait, which is then pressed to the fibrous layer on the conveyor belt.

[0020] The device is suitable for the production of a fibrous layer in which the fibres are oriented predominantly perpendicular towards the fabric plane.

Example 2

[0021] A device shown in Fig. 2 consists of two vibrating elements 1 and 2 as in Fig. 1.

[0022] The element 1 is connected with a driving mechanism 3 over a shaft 4 and the element 2 over a shaft 5 by means of tie-rods 7 with sliding fitting and flexible knuckle joints. The driving mechanism 3 shown in Fig. 2a consists of a driving shaft 8 with two crank shafts 9. The crank shafts are set up to allow advanced phase movement of one of vibrating elements.

[0023] In comparison with Example 1, due to the advanced phase movement of vibrating element 2, the processed fiber layer is better taken off the vibrating element 1. Therefore the folds are more regular and the fabric of smooth surface is produced.

Example 3

[0024] A device in Fig. 3 consists of the same elements as in Example 2, both the vibrating elements 1 and 2 are linked with shafts 4 and 5 by means of tie-rods 6 and 7 with sliding fitting and flexible knuckle joints.

[0025] The advantage of the device is the straight-lined movement of both vibrating elements 1 and 2 which does not cause air turbulence and vibrations of the fed in carded web. It leads to improved regularity of final fabric.

Example 4

[0026] A device as in Example 3, the flexible knuckle joints 6 and 7 are replaced with steel springs. Due to this the mass of links and dynamic loading of driving mechanism is reduced and the life of device improved.

Utilization of the device

[0027] The device for perpendicular stratification of planary fibrous shapes is utilizable namely in the textile industry.

Claims

1. A device for perpendicular stratification of planary fibrous shapes, above all a fibrous web, with two elements making synchronous and reciprocating motions and connected with a driving mechanism, **characterized in that** the elements (1,2) exerting a reciprocating motion are connected with the driving mechanism (3) indirectly over at least one robust shaft (4) rigidly fitted in bearings in a rigid framework of the machine, while one element (1) is coupled to the shaft (4) rigidly or over flexible joints (6), and a second element (2) is coupled by means of flexible joints (7) with the same shaft or with another shaft (5).
2. A device according to Claim 1, **characterized in that** the driving mechanism (3) consists of one driving shaft (8) with two crank assemblies (9) arranged with a phase shift to each other.
3. A device according to Claims 1 and 2 **characterized in that** the flexible joints (6) and (7) consist of flat steel springs with width-to thickness ratio more than 10.

Patentansprüche

1. Vorrichtung zur senkrechten Schichtung von ebenen Faserformen, insbesondere einer Faserbahn, mit zwei Elementen, die synchrone und wechselnde Bewegungen ausführen und mit einem Antriebsmechanismus verbunden sind, **dadurch gekennzeichnet, dass** die eine wechselnde Bewegung ausführenden Elemente (1, 2) mit dem Antriebsmechanismus (3) indirekt über mindestens eine starke Welle (4) verbunden sind, die spielfrei in Lagern eines ortsfesten Rahmens der Maschine gelagert sind, wobei ein Element (1) mit der Welle (4) fest oder über flexible Verbindungen (6) verbunden ist und ein zweites Element mittels flexibler Verbindungen (7) mit derselben Welle oder mit einer weiteren Welle (5) verbunden ist.
2. Vorrichtung nach Anspruch 1, **dadurch gekennzeichnet, dass** der Antriebsmechanismus (3) eine Antriebswelle (8) mit zwei Kurbelwellen (9) umfasst, die gegeneinander phasenverschoben sind.
3. Vorrichtung nach den Ansprüchen 1 und 2, **dadurch gekennzeichnet, dass** die flexiblen Verbindungen (6, 7) aus flachen Stahlfedern mit einem Breite-Stärke-Verhältnis von mehr als 10 bestehen.

Revendications

1. Dispositif pour une disposition en couches perpendiculaires des formations filamenteuses planes, en particulier de la voile de cardé, avec deux éléments mobiles effectuant des mouvements synchrones et réversibles, raccordés au dispositif de propulsion, **caractérisés par** le raccordement indirect des éléments avec le mouvement réversible (1, 2) au dispositif de propulsion (3) par l'intermédiaire d'au moins un axe massif (4) monté fermement dans les paliers dans le cadre massif de la machine, un élément (1) étant couplé avec l'axe (4) fermement ou par l'intermédiaire des joints flexibles (6) et l'autre élément (2) étant couplé, par des joints flexibles (7), avec le même ou un autre axe (5).
2. Le dispositif d'après l'exigence 1, **caractérisé par** la propriété, que le dispositif de propulsion (3) est formé par un axe de propulsion (8) avec deux mécanismes à manivelle (9) ajustés avec une différence mutuelle de marche.
3. Le dispositif d'après l'exigence 1 et 2, **caractérisé par le fait que** les raccordements flexibles (6) et (7) sont formés par des ressorts plats en acier avec un

rapport entre la largeur et l'épaisseur plus grand que 10.

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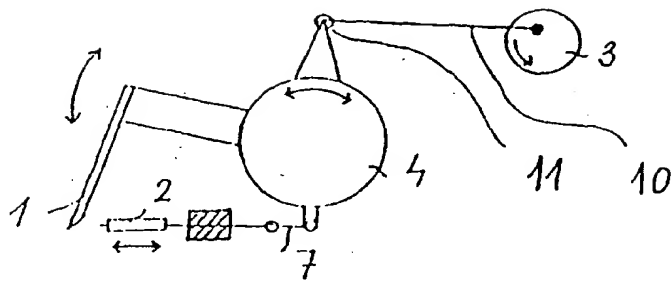


Fig. 1

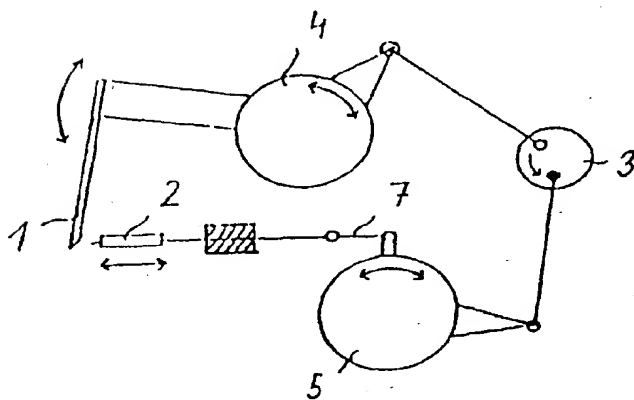


Fig. 2

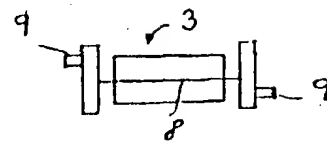


Fig. 2a

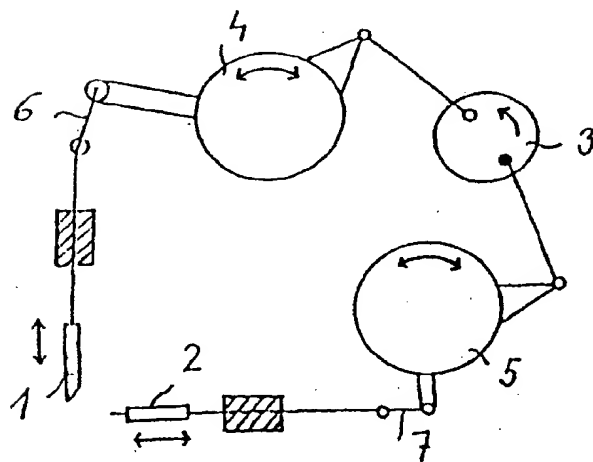


Fig. 3

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